

**IN THE UNITED STATES
PATENT AND TRADEMARK OFFICE**

TITLE:

**Manual Brake for a Wheelchair with a
Variable Braking Force**

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BACKGROUND OF THE INVENTION

This application is a continuation in part of my now pending U.S. Patent Application Serial No. 10/622,339 filed on July 18, 2003.

Field of The Invention

The present invention relates generally to the field of wheelchairs and, more specifically, to a manual braking system with a variable braking force and quick release, detachable wheels for manual wheelchairs.

Description of the Related Prior Arts

Numerous types of braking mechanisms for manual wheelchairs are known in the art. The most typical manual wheelchair brake is a manual "over center" locking device which is activated by a lever arm and, when forced into its locking position, presses a braking member against the surface of the wheelchair tire creating a frictional braking action. Several factors mitigate against the usefulness and reliability of these types of brakes. Loss of tire pressure reduces the frictional force exerted by the crossbar on the tire and hence reduces the braking effect. A significant air pressure loss leaves these brakes useless. During transfer in and out of the chair, this type of brake allows the tire to slide underneath the crossbar and the wheelchair to move. Similarly, the brakes are ineffective and will not adequately hold the wheelchair on an

1 incline. Other types of manual brakes include caliper type brakes manually activated with a
2 lever arm mounted to a cable and brake assembly causing brake pads to press against the rim
3 of the wheelchair wheel. Typically, braking mechanisms for wheelchairs only apply a braking
4 force to one wheel. If an equal braking force is desired on both wheels, the user is required to
5 perform the difficult task of using both arms at the same time. Finally, these types of manual
6 brakes, whether caliper type brake or not, do not allow for a variable braking force to be
7 exerted on the tire or rim. A variable braking force allows the user to both slow the wheelchair
8 and ultimately stop it and hold it in place when desired.

9 Patents to Ross and Gunther, U.S. Patent No. 5,358,266 and Lautzenhiber, U.S.
10 Patent No. 4,805,711 describe a braking member, which applies a braking frictional force
11 directly to the wheelchair tire which is manually activated by a lever arm. There are also
12 disclosed in the art several manual braking mechanisms which utilize a cable actuated caliper
13 braking mechanism on the rim of one wheel or on the rims of both wheels with two distinct
14 braking systems operating separately. Examples of these types of braking mechanisms are
15 disclosed in patents to Herron, U.S. Patent No. 4,560,181; Kaweck, U.S. Patent No.
16 4,204,588; and Lemarie, U.S. Patent No. 4,538,826. Finally, a patent to Berry, U.S. Patent
17 No. 5,492,355 discloses a caliper type braking mechanisms that discloses caliper type brakes
18 which operate on the tire rim of each wheelchair wheel and can be activated by the use on one

1 lever. Many of the same deficiencies discussed above apply to each of these braking
2 mechanisms.

3 Wheelchair users have reason to frequently remove the wheels from their wheelchairs.
4 It is often done for storage purposes, for brake adjustment, for wheel repair, and for wheel
5 exchange. For example, in order to store a wheelchair in a vehicle, it is often desirable to
6 remove the wheels.

7 Heretofore, the wheels on manual wheelchairs and other types of wheelchairs have
8 been attached to the wheelchair frame by some type of hub with the wheels secured to the hub
9 with nuts and bolts. In order to remove the wheels from the wheelchair, it has been necessary
10 to unscrew and remove each of the nuts and bolts securing the wheel to the hub. This is a time
11 consuming and cumbersome process. Once again, wheelchair users who have arm or hand
12 limitations may not be physically able to remove the nuts and bolts.

13 More recently, it has become common in the art to attach wheels to manual wheelchairs
14 using quick release locking pins which hold the wheel to the axle. In this type of design, it is
15 difficult to also have a braking means on the wheelchair wheel other than the manual "over
16 center" locking device which presses a braking member against the surface of the tire as
17 described herein. Heretofore, other braking systems such as those which utilize caliper type
18 brakes operating on the rim of the wheelchair wheel, have been ineffective on wheelchairs with

1 quick release locking pins because the braking means had to be released and moved or
2 disassembled in order to remove the wheel and thereby defeating the purpose of the quick
3 release locking pin.

4 It is desirable to have a wheelchair with an effective easily operatable manual braking
5 mechanism and, at the same time having quick release detachable wheels.

6 7 SUMMARY OF THE INVENTION

8 It is an object of this invention to provide a manually activated braking system for a
9 wheelchair which provides a braking force to a disk, as opposed to the tire surface or rim of the
10 wheelchair wheel and thereby provide more efficient braking action.

11 It is a further object of this invention to provide a manual braking system for a
12 wheelchair which allows for a variable braking force to slow the wheelchair during operation.

13 It is a further object of this invention to provide a braking system for manual
14 wheelchairs, which provides equal braking force to both wheels of a wheelchair simultaneously.

15 It is a further object of this invention to provide a manual braking means for manual
16 wheelchairs, which allows for detaching the wheelchair wheels without disturbing the braking
17 means.

18 It is a further object of this invention to provide for quick release, easily detachable

1 wheels.

2 It is a further object of this invention to provide for detachable wheels, which eliminates
3 the need for users of the wheelchair to unscrew numerous nut and bolt combinations in order to
4 remove the wheel.

5 It is a further object of this invention to provide for quick release, easily detachable
6 wheels which allow the wheels to be removed without removing the disk and brake assembly.

7 In order to achieve these objectives, this invention provides for a manual braking
8 system for a wheelchair which is comprised of a braking means, a cable pully system attached
9 to the braking means, and a manual lever assembly pivotally mounted to the wheelchair frame
10 for activating the braking means.

11 It is anticipated that the preferred braking means is a caliper-type brake positioned to
12 clamp onto a metal disk mounted axially to a hub which rotates on the axle of each wheelchair
13 wheel. The hub on which the disk is mounted interlocks with the hub on which the wheelchair
14 wheel is mounted. The interlocking hubs are locked together with a locking pin, which extends
15 axially through the center of the mated hubs such that the hubs are locked and rotate together
16 when the wheelchair wheel is turned.

17 The locking pin is equipped with retractable nipples which, when extended, hold the
18 locking pin securely in place. The retractable nipples are spring biased in the extended position

1 and are activated by a push button at one end of the locking pin which releases the spring and
2 allows the nipples to retract. When the nipples are in the retracted position, the locking pin can
3 be removed simply by sliding it out of the axle. This allows the wheelchair wheel to be
4 removed since there is no longer anything holding the mated hubs together.

5 The manual lever assembly comprises a mounting bracket having a mounting shoulder,
6 a lever arm pivotally attached to the mounting bracket, and a linking member pivotally attached
7 to both the lever arm and the pulley. The linking member is attached in such a manner that
8 when the lever arm is rotated, the linking member is displaced horizontally and thereby causing
9 horizontal displacement of the pulley.

10 The braking means for each wheel are connected to opposite ends of a cable wire.
11 The ends of the cable wire are directed through small openings in perpendicular element of the
12 mounting bracket and around the pulley such that displacement of the pulley provides equal
13 force and displacement to said opposite ends of the cable wire. The small openings are spaced
14 a distance equal to the diameter of the pulley so the cable wire remains parallel as it extends
15 from the pulley through said openings. The pulley is axially and pivotally connected to the
16 linking member and positioned between the mounting bracket and the linking member. A pin
17 connecting the pulley and the linking member also extends through and slides in a horizontal slot
18 in the mounting bracket and thereby causing the displacement of the pulley to be in a horizontal

1 plane.

2 The lever arm can be rotated in two different directions. When the actuating lever is
3 rotated in a first direction, it will cause the linking member to displace horizontally pulling the
4 pulley and cable wires and activating the braking force. The further the lever arm is rotated, the
5 greater the braking force exerted on the disk and the operator can vary the braking for in this
6 manner. When the lever arm is rotated in the opposite direction, it will cause the linking member
7 to displace in the opposite horizontal direction pushing the pulley and cable wire and
8 deactivating the braking force.

9 I an alternate embodiment of this invention, a plunger assembly with a spring biased
10 rounded head is mounted through an opening in the mounting bracket. The plunger assembly is
11 positioned to allow the rounded head to extend into a series of semi-hemispherical notches on
12 the inner surface of the actuating lever. The notches are radially spaced around the pin
13 connecting the lever arm to the mounting bracket. The notches are positioned such that each
14 notch will separately receive the plunger pin as the lever arm is rotated. The force exerted by
15 the spring and the plunger pin is sufficient to prevent the lever arm from rotating until it receives
16 sufficient manual force. In this manner, each notch represents a different level of braking force
17 to be applied to the disk.
18

BRIEF DESCRIPTION OF THE DRAWINGS

Fig. 1 is a elevational side view of a manual wheelchair depicting manual brake actuator assembly and a caliper braking mechanism mounted to the wheelchair frame and positioned to clamp onto a metal disk mounted axially to the hub of the wheelchair wheel.

Fig. 2A is an enlarged exploded perspective view depicting the locking pin, wheelchair wheel, hub, disk, and axle assembly which has a spring biased push button type locking pin and first interlocking hub design.

Fig. 2B is an enlarged exploded perspective view depicting the locking pin, wheelchair wheel, hub, disk, and axle assembly wherein the locking pin is equipped with a lever which activates an expandable tip.

Fig. 2C is an enlarged exploded perspective view depicting Fig. 2A from the opposite angle.

Fig. 2D is an enlarged exploded perspective view depicting the locking pin, wheelchair wheel, hub, disk, and axle assembly. This figure depicts a second interlocking hub design.

Fig. 2E is an enlarged exploded perspective view depicting Fig. 2D from the opposite angle.

Fig. 3 is a bottom view of the wheelchair seat depicting the manual brake actuator assembly mounted to the wheelchair frame.

1 Fig. 4 is a rear elevational view depicting the clamp and mounting bracket of the manual
2 brake actuator assembly.

3 Fig. 5 is a perspective view depicting the caliper brake mechanism and disc.

4 Fig. 6 is an exploded perspective view depicting the manual brake actuator assembly.

5 Fig. 7 is a perspective view depicting the manual brake actuator assembly.

6 Fig. 8A is a perspective view of the manual brake actuator assembly depicting the lever
7 arm is a vertical non-braking position.

8 Fig. 8B depicts the manual brake actuator assembly with the lever arm in a partially
9 braking position.

10 Fig. 8C depicts the manual brake actuator assembly with the lever arm in a horizontal
11 full braking position.

12 Fig. 9A is a side view of an alternative embodiment of the present invention depicting
13 the manual brake actuator assembly with a plunger mechanism with the lever arm in a vertical
14 non-braking position.

15 Fig. 9B is the alternative embodiment of the present invention as shown in Fig. 9A with
16 the actuating lever in a partial braking position.

17 Fig. 9C is the alternative embodiment of the present invention as shown in Fig. 9A and
18 9B with the actuating lever in a horizontal full braking position.

1 Fig. 10 is an enlarged perspective view of the plunger mechanism provided in the
2 alternative embodiment of the present invention depicted in Figs. 9A-9C.

3 4 **DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENT**

5 Referring to Fig. 1, a lightweight manual wheelchair 10 is equipped with a seat 12, and
6 seat back 13 mounted between first and second wheelchair wheels 24 generally to a frame 14.
7 The frame 14 has a vertical component 15, a side horizontal component 16, a frontal curved
8 component 17 and a lower curved component 20. A footrest 19 is mounted at the frontal
9 extremity of the lower curved component 20 of the frame 14. First and second caster wheels
10 21 are pivotally mounted toward the frontal extremity of the lower curved component 20 of the
11 frame 14. The manual wheelchair 10 is symmetrical about a centre line and the opposed side is
12 identical to the side visible in Fig. 1. Thus, when the first and second of numbered items are
13 referred to without the second item being shown, it can be appreciated that the second
14 numbered item is identical to the first but on the opposite side of the wheelchair.

15 Referring to Figs. 2A through 2E, the first and second disks 22 are concentrically
16 mounted to the inner face 83 first and second disk hubs 23 by means of a plurality of screws
17 29 passing through radially spaced interiorly threaded, aligned holes 51 in the first and second
18 disk hubs 23 and the first and second disks 22. In the preferred embodiment, as shown in Figs.

1 2A, 2B, and 2C, the screws 29 are Allen screws where the heads 33 of the screws 29 extend
2 from the outer vertical faces 27 of the first and second disk hubs 23 and are secured on the
3 opposite end by nuts 38. In a second preferred embodiment, as shown in Figs. 2D and 2E, the
4 screws 29 are of a length insufficient to extend beyond the outer vertical faces 27 of the first
5 and second disk hubs 23.

6 The first and second disk hub 23 and disk 22 assemblies are concentrically mounted to
7 outer ends of first and second detachable axle pieces 80 and rotate thereon. The first and
8 second detachable axle pieces 80 are tubular with a smooth surface portion 82 at their outer
9 end and a exteriorly threaded portion 84 at their inner end. The smooth surface portion 82 and
10 the exteriorly threaded portion 84 are divided by a flange 86.

11 The first and second detachable axle pieces 80 are mounted to the frame 14 of the
12 wheelchair 10 (see Fig. 1) by screwing the exteriorly threaded portion 84 into a tubular axle 25.
13 As shown in Fig. 3, the tubular axle 25 is clamped to the first and second lower curved
14 components 20 of the frame 14 (See Fig. 1) at its rear extremity by first and second frame
15 clamps 72.

16 Referring again to Figs. 2A through 2E, the outer ends of the tubular axle 25 have
17 mounting heads 88. Each mounting head 88 has a threaded bore 90 with a diameter sufficient
18 to accept and secure the exteriorly threaded portion 84 of the first and second detachable axle

1 pieces 80 therein. The first and second detachable axle pieces 80 are mounted to the tubular
2 axle 25 by screwing the exteriorly threaded portion 84 into the threaded bore 90.

3 The first and second disk hub 23 and disk 22 assemblies are secured to the first and
4 second detachable axle pieces 80 by means of a clip ring 39. The clip ring 39 is spring biased
5 to close around and fit in to a circumferential groove 78 cut into the smooth surface portion 82
6 of the first and second detachable axle pieces 80 at their extreme outer end. In order to allow
7 the first and second disk hub 23 and disk 22 assemblies to rotate on the first and second
8 detachable axle pieces 80, the smooth surface portion 82 of the first and second detachable
9 axle pieces 80 extend axially through a tubular opening 92 at the center of the first and second
10 disk hubs 23 and the outer face of flange 86 abuts a concentric circular shoulder 87 (see Figs.
11 2C and 2D) on the inner face 83 of the first and second disk hubs 23 with a spacer ring 94
12 between. The spacer ring 94 prevents frictional contact between the outer face of flange 86
13 and the circular shoulder 87 on the inner face of the first and second disk hubs 23. In the
14 preferred embodiment, the spacer ring 94 is a Delrin washer. However it is anticipated that
15 other smooth, durable material can be substituted.

16 Referring to Figs. 2A, 2B, and 2E, the outer vertical face 27 of the first and second
17 disk hub have a concentric circular recessed portion 93 surrounding the tubular opening 92.
18 The horizontal length of the smooth surface portion 82 of the detachable axle piece 80 is

1 sufficient to allow the smooth surface portion 82 to extend through the tubular opening 92 of the
2 first and second disk hubs 23 and expose the circumferential groove 78 on the opposite side of
3 the first and second disk hubs 23 with minimal clearance at the concentric circular recessed
4 portion 93. This allows the clip ring 39 to close around circumferential groove 78 within the
5 concentric circular recessed portion 93.

6 As shown in Figs. 2A through 2C, the first and second wheelchair wheels 24 are
7 concentrically mounted on the first and second wheel hubs 37. The inner surface 57 of the first
8 and second wheelchair wheels 24 (See Fig. 2C) is mounted flush against the outer vertical
9 surface 70 (See Fig. 2E) of the flanged inner portion 31 of the first and second wheel hubs 37
10 and are secured to the first and second wheel hubs 37 by first and second nuts 45, which screw
11 onto exteriorly threaded outer ends 75 of the first and second wheel hubs 37. The first and
12 second wheel hubs 37 have a tubular opening 43 through their center. As shown in Figs. 2A
13 and 2B, an outer circular bearing assembly 61 is pressed fit into the tubular opening 43 towards
14 the outer end of the first and second wheel hubs 37.

15 As shown in Figs. 2B, 2C, and 2D, an inner circular bearing assembly 79 is pressed fit
16 into the tubular opening 43 at the inner end of the first and second wheel hubs 37. The outer
17 bearing assembly 61 and inner bearing assembly 79 have inner rings 63 which turn within the
18 bearing assemblies. The inner diameter of the inner rings 63 is equal to the inner diameter of first

1 and second detachable axle pieces 80. In the preferred embodiment, the outer circular bearing
2 assembly 61 and inner circular bearing assembly 79 are manufactured by NICE, Model No.
3 1616 DC TN or KYK, Model No. R-8-DDHA1(IB). However, it is anticipated that other
4 similar bearings could be used.

5 Referring again to Figs. 2A through 2E, when the first and second wheelchair wheels 24
6 are mounted to the wheel hub 37 and in turn mounted to the wheelchair 10 (See Fig. 1), the
7 outer vertical faces 27 of the first and second disk hubs 23 interlock with inner faces 77 of the
8 flanged inner portion 31 of the first and second wheel hubs 37. In the preferred embodiment,
9 as shown in Figs. 2A, 2B, and 2C, the inner faces 77 of the flanged inner portion 31 of the first
10 and second wheel hubs 37 are flat with a plurality of radially spaced holes 96 shown in Fig. 2C.
11 The heads 33 of the plurality of screws 29 fit snugly into the corresponding radially spaced
12 circular holes 96 in the flanged inner portion 31 of the first and second wheel hubs 37. In an
13 alternate embodiment, as shown in Figs. 2D and 2E, the inner face 77 of the flanged inner
14 portion 31 of the first and second wheel hubs 37 have a raised surface 98 extending from the
15 inner face 77. The raised surface 98 is centered on the inner face 77 with parallel sides 100
16 extending to the circumference of the inner face 77. The parallel sides 100 extend
17 perpendicularly from the inner face. In this alternate embodiment, the outer vertical faces 27 of
18 the first and second disk hubs 23 have a channel 102. The placement and dimensions of the

1 channel 102 are to allow the raised surface 98 to fit snugly into the channel 102 with minimal
2 clearance at all contiguous surfaces when the first and second wheel hubs 37 are interlocked
3 with the first and second disk hubs 23.

4 In the preferred embodiment, as shown in Figs. 2A, 2B, and 2C, the interlocking of
5 heads 33 within the radially spaced circular holes 96 cause the first and second wheelchair
6 wheels 24 and the first and second disks 22 to rotate together. In another alternate
7 embodiment, as shown in Figs. 2D and 2E, the interlocking of the raised surface 98 on the inner
8 face 77 of the first and second wheel hubs 37 with the channel 102 in the outer vertical faces 27
9 of the first and second disk hubs 23 cause the first and second wheelchair wheels 24 (See Fig.
10 1) and the first and second disks 22 to rotate together.

11 Still referring to Figs. 2A through 2E, in order to hold the first and second disk hubs
12 and the first and second wheel hubs together when interlocked, first or second locking pins 35a
13 and 35b (see Figures 2A and 2B) extend axially through the center of the first and second
14 wheel hubs 37, the first and second disk hubs 23, and into the first and second detachable axle
15 pieces 80. The first or second locking pins 35a and 35b have a diameter which allows the first
16 or second locking pins 35a and 35b to slide through the inner rings 63 of the outer circular
17 bearing assembly 61 (See Figs. 2A and 2B) and the inner circular bearing assembly 79 (See
18 Figs. 2C and 2D) and into the first and second detachable axle pieces 80 with minimal

1 clearance.

2 The first and second wheelchair wheels 24 can be detached from the wheelchair 10
3 (See Fig. 1) without removing the first and second disks 22 or disturbing the first and second
4 caliper brakes 18 by removing the first and second locking pins 35a or 35b and separating the
5 first and second wheel hubs 37 from the first and second disk hubs 23.

6 In the preferred embodiment of the invention (see Figures 2A, 2C, 2D, and 2E), the
7 first and second locking pins 35a have a push button 47, a rod 49, an adjusting nut 53, and a
8 set of retractable nipples 55. The push button 47 is spring biased in the released position,
9 causing the retractable nipples 55 to extend from the rod 49. When the push button 47 is
10 depressed, the retractable nipples 55 retract into the rod 49. The first and second locking pins
11 35a can be inserted through the inner ring 63 of the outer circular bearing assembly 61 and into
12 the tubular openings 43 of the first and second wheel hubs 37 by depressing the push button 47
13 and thereby causing the retractable nipples 55 to retract. When the first and second locking
14 pins 35a are further inserted through the first and second disk hubs 23 and into the first and
15 second detachable axle pieces 80 and the push button 47 is released, the retractable nipples 55
16 extend into grooves (not shown) circumferentially cut into the tubular interior surface (not
17 shown) of the first and second detachable axle piece 80. The grooves (not shown) are of
18 sufficient depth and width to allow the retractable nipples 55 to extend into the grooves (not

1 shown) with minimal clearance. The grooves (not shown) are positioned in the first and second
2 detachable axle pieces 80 to allow the retractable nipples 55 to extend into the first and second
3 grooves (not shown) when the first and second locking pins 35a are fully inserted into the first
4 and second wheel hubs 37 such that the adjustable nut 53 contacts the outer surface of the
5 outer circular bearing assembly 61. In the preferred embodiment, the first and second locking
6 pins 35a are QRP Quick Release Push Button (large/small) Axle, Model No.
7 21QRP11CDASN.

8 In an alternate embodiment of the invention, the length of the exteriorly threaded portion
9 84 of the first and second detachable axle pieces 80 is sufficient to allow the position of the
10 retractable nipples 55 on the first and second locking pins 35a to extend beyond the inner lip 85
11 of the first and second detachable axle pieces 80 when the first and second locking pins 35a are
12 fully inserted into the first and second wheel hubs 37 such that the adjustable nut 53 contacts
13 the outer surface of the outer circular bearing assembly 61. Thus, when the first and second
14 locking pins 35a are fully inserted and the push button 47 is released, the retractable nipples 55
15 extend adjacent to the inner lip 85 of the first and second detachable axle pieces 80 with
16 minimal clearance and thereby holding the first and second locking pins 35a in place. In this
17 embodiment, the first and second locking pins 35a are, once again, QRP, Quick Release Push
18 Button (large/small), Axle Model No. 21QRP11CDASN.

1 In yet another embodiment of the invention (see Figure 2B), the first and second locking
2 pins 35b have a release lever 65 at one end of a rod 67, a spacer joint 69 between the release
3 lever 65 and the rod 67, an expandable tip 71 attached to the other end of the rod 67, and a
4 wedging cap 73 attached to the expandable tip 71 opposite the rod 67. When the release lever
5 65 is rotated to the released position so that it extends parallel with the rod 67, the diameter of
6 the expandable tip 71 is not expanded and is equal to the diameter of the rod 67. When the
7 release lever 65 is rotated perpendicular to the rod 67, the wedging cap 73 is pulled toward the
8 release lever 65 causing the expandable tip 71 to expand to a diameter greater than the
9 diameter of the rod 67. When the release lever 65 is in the released position, the first and
10 second locking pins 35b can be inserted through the inner ring 63 of the outer circular bearing
11 assembly 61 and into the tubular opening 43 of the first and second wheel hubs 37. When the
12 first and second locking pins 35b are inserted through the first and second wheel hubs 37, and
13 into the first and second detachable axle pieces 80 and the release lever 65 is then rotated
14 perpendicular to the rod 67, the expandable tip 71 expands into and makes frictional contact
15 with the interior surface (not shown) of the first and second detachable axle pieces 80. The
16 frictional force created is great enough to hold the first and second locking pins 35b in place.
17 The diameter of the spacer joint 69 is greater than the inner diameter of the inner ring 63 of the
18 outer circular bearing assembly 61, such that when the first and second locking pins 35b are

1 fully inserted, the spacer joint 69 contacts the outer face of the outer circular bearing assembly
2 61. In this preferred embodiment, the locking pin 35b is the Ultra Axle, 0.50" O.D.
3 manufactured by Rousson Chamoux.

4 Referring to Figs. 1, 3, 4, and 6, a manual brake actuator assembly 120 has a mounting
5 bracket 122 which is fixed to the horizontal portion 16 of the frame 14 of the wheelchair 10 by
6 a semi-circular mounting shoulder 124 and a clamp 126. The clamp 126 has an upper element
7 128 and a lower element 130 which when clamped together with a screw 127, form a first
8 channel 129a and a second channel 129b at opposing ends of the clamp 126. The first channel
9 129a and second channel 129b have curved interior surfaces (not shown). The radial
10 dimensions of the curved interior surfaces (not shown) of the first channel 129a and second
11 channel 129b are sufficient to allow the first channel 129a and second channel 129b to engage
12 and clamp onto the horizontal portion 16 of the frame 124 and the mounting shoulder 124
13 respectively when the upper element 128 and the lower element 130 of the clamp 126 are
14 clamped together. As shown in Fig. 6, the mounting shoulder 124 is mounted to a generally
15 rectangular base portion 132 of the mounting bracket 122 with flathead screws 134.

16 Referring to Figs. 6, 7, and 8a through 8c, the mounting bracket 122 has a straight,
17 horizontal lower edge 136 and gradually narrows along its horizontal length from the base
18 portion 132 at rearward end to a rounded tip 138 at forward end. A perpendicular element

1 140 which is generally rectangular in shape, extends perpendicularly and outward from the base
2 portion 132 of the at its rearward end.

3 Still referring to Figs. 6, 7, and 8a through 8c, an elongated actuating lever 142 is
4 pivotally mounted to the mounting bracket 122 with first allenhead screw 144 having a
5 cylindrical head 146, an intermediate smooth portion 148 and a threaded portion 150. The first
6 allenhead screw 144 is inserted through a non-threaded hole 152 in the actuating lever 142 and
7 into a threaded hole 154 in the mounting bracket 122 such that the actuating arm 142 can pivot
8 on the smooth portion 148 of the first allenhead screw 144.

9 As shown in Fig. 6, a first protecting sleeve 156 is inserted in the non-threaded hole
10 152 of the actuating lever 142 around fist the allenhead screw 144. A first washer 158 is
11 axially mounted on the first allenhead screw 144 between the head 146 and the outer surface of
12 the actuating lever 142. A second washer 160 is axially mounted on the first allenhead screw
13 144 between the inner surface of the actuating lever 142 and the outer surface of the mounting
14 bracket 124.

15 Referring again to Figs. 6, 7, and 8a through 8c, a linking element 162, having a circular
16 rearward portion 188, a circular forward portion 176 and a bridging member 177 extending
17 between the rearward and forward portions, is pivotally connected to the actuating lever 142
18 with a second allenhead screw 164 having a head 166, an intermediate smooth portion 168,

1 and a threaded portion 170. The second allenhead screw 164 extends through a second non-
2 threaded hole 172 in the actuating lever 142 and then through a threaded hole 173 in the center
3 of the forward end 176 of the linking element 162. The second allenhead screw is positioned
4 such that the head extends from the inner surface of the actuating lever 142 and the rounded
5 surface of the head engages and rides on the rounded surface 177 of the rounded tip 138 of the
6 mounting bracket 122 as the actuating lever 142 is rotated. A third washer 178 is axially
7 mounted on the second allenhead screw 164 between the head 166 and the inner surface of the
8 actuating lever 142. A fourth washer 180 is axially mounted to the second allenhead screw 164
9 between the outer surface of the actuating lever 142 and the inner surface of the linking element
10 162. A second protective sleeve 182 is inserted into the second non-threaded hole 172 of the
11 actuating lever 142 around the second allenhead screw 162 and pulley 184.

12 Still referring to Figs. 6, 7, and 8a through 8c, a pulley 184 is axially and pivotally
13 mounted between the mounting bracket 122 and the rearward end 188 of the linking element
14 with a third allenhead screw 186. The third allenhead screw 186 is inserted through a
15 horizontal guiding slot 190 cut in the mounting bracket 122. The third allenhead screw 186 has
16 a head 192, intermediate smooth portion 192, and a threaded portion 196. The third allenhead
17 screw 186 is positioned such that the head 192 extends from the inner surface of the mounting
18 bracket 122; the intermediate smooth portion 194 extends through the guiding slot 190 and an

1 axial hole 198 in the pulley 184; and the threaded portion 196 extending into a threaded hole
2 200 in the center of the rearward end 188 of the linking element 162.

3 Referring to Fig. 6, a fifth washer 202 is axially mounted on the third allenhead screw
4 186 between the head 192 and the inner surface of the mounting bracket 122. A sixth washer
5 204 is axially mounted from the third allenhead screw 186 between the outer surface of the
6 mounting bracket 122 and the inner surface of the pulley 184. A seventh washer 206 is axially
7 mounted on the third allenhead screw 186 between the outer surface of the pulley 184 and the
8 inner surface of the linking element 162. The protective sleeve 208 is inserted in the axial hole
9 198 of the pulley 184 around the third allenhead screw 186.

10 As shown in Fig. 7, the guiding slot 190 is generally rectangular in shape and elongated
11 horizontally. The horizontal centerline of the guiding slot is horizontally aligned with the
12 centerline of the threaded hole 154 in the mounting bracket 122. The inner face of the
13 mounting bracket 122 has a recessed ledge 210 which surrounds the guiding slot 190. The
14 vertical width of the recessed ledge 210 around the guiding slot 190 sufficient to allow fifth
15 washer and head 192 of the third allenhead screw 186 to fit between and upper lip 212 and a
16 lower lip 214 of the recessed ledge 210.

17 Referring to Figs. 1 and 5, first and second caliper brakes 18 are mounted to extension
18 plates (not shown) which are in turn mounted to the frame 14 of the wheelchair 10. The caliper

1 brakes 18 are positioned to clamp onto first and second disks 22. In the preferred
2 embodiment of this invention, the first and second caliper brakes 18 are manufactured by
3 Hayes/HMX, model number BR3920. However, numerous other cable actuated caliper
4 brakes are available on the market and can be used in this invention. The first and second
5 wheelchair wheels 24 can be detached without removal of the first and second disks 22 or the
6 first and second caliper brakes 18.

7 Still referring to Figs. 1 and 5, the first and second caliper brakes 18 are activated by
8 pulling a cable wire 110 (See Figs. 4 and 5) attached to the caliper brakes 18 at first and
9 second ends of the cable wire 110. The first and second ends of the cable wire 110 are
10 directed to the first and second caliper brakes 18 through a cable wire housing 112 which is
11 attached to a nozzle 114 on the first and second caliper brakes 18. The first and second ends
12 of the cable wire 110 are attached to the first and second caliper brakes 18, respectively, in
13 typical fashion. The cable wire 110 passes through the nozzle 114 of the first and second
14 caliper brakes 18 and into the cable wire housing 112.

15 Referring now to Figs. 1, 7, and 8a through 8c, the cable wire 110 is directed from the
16 first and second caliper brakes 18 through the cable wire housing 112 to the manual actuating
17 brake assembly. The cable wire 110 extends to through small openings 116 and around the
18 pulley 184 in the perpendicular element 140 of the mounting bracket 122. The centers of the

1 small openings 216 are equal distance of the base portion 132 of the mounting bracket 122 and
2 are vertically spaced a distance equal to the diameter of the pulley 184.

3 Referring to Figs. 1 and 8a through 8c, in operation of preferred embodiment of this
4 invention, the first and second caliper brakes 18 are activated by rotating the actuating lever
5 142. When the actuating lever 142 is in its upright, vertical positioned as shown in Fig. 8a, the
6 first and second caliper brakes 18 are deactivated. As the actuating lever 142 is rotated in a
7 forward direction as shown in Fig. 8b, it causes the linking element 162 to rotate and at the
8 same time displace in a forward horizontal direction. The linking element 162, in turn, causes
9 the pulley 184 to displace in a forward, horizontal direction. As the pulley 184 displaces
10 forward, the intermediate smooth portion 194 of the third allenhead screw 186 slides forward
11 within the guiding slot 190 and thereby maintaining the movement of the pulley 184 in constant
12 horizontal plane. Additionally, as the pulley 184 moves in a forward direction, it pulls the cable
13 wire 110 and thereby activating the first and second caliper brakes 18 with equal force. The
14 first and second caliper brakes 18 are released by rotating the actuating lever 142 backward
15 towards its vertical, upright position as shown in Fig. 8a.

16 As shown in Figs. 8a and 8c, as the actuating lever 162 is rotated forward, the curved
17 surface 218 of the forward end 176 of the linking element 162 engages and rides on the curved
18 surface of the head 146 of the first allenhead screw 144. When the actuating lever 142 is

1 rotated forward to a horizontal position, as shown in Fig. 8c, the linking element 162 moves to
2 an “overcenter” locking position such that the curved surface 218 of the forward end 176 of the
3 linking element 162 engages the curved surface of the head 146 of the first allenhead screw 144
4 at its forward most point. In this position, the maximum braking force of the caliper brake 18 is
5 achieved and prevents further movement of the wheelchair.

6 An alternate embodiment of the invention is shown in Figs. 9a through 9c and Fig. 10.
7 In this embodiment, a exteriorly threaded plunger assembly 220 extends through the mounting
8 bracket 122 through an interiorly threaded hole (not shown) and is secured to the mounting
9 bracket 122 with a nut 224. The plunger assembly 220 has a spring biased rounded plunger
10 head 226. The plunger assembly 220 is positioned to allow the plunger head 226 to extend
11 into a plurality of semi-hemispherical notches 228 on the inner surface of the actuating lever
12 142.

13 The notches 228 are radially spaced around the first non-threaded hole 152 in the
14 actuating lever 142. The notches are positioned to receive the plunger head 226 when the
15 actuating lever 142 is rotated to a series of positions equal to the number of notches 228. The
16 first in the series of notches 228 is positioned to the plunger head 226 when the actuating lever
17 142 when it is in an upright vertical position, as shown in Fig. 9a, and the braking force is
18 deactivated. The last in the series of notches 228 is positioned to receive the plunger head 226

1 when the actuating lever 142 in the horizontal position, as shown in Fig. 9c, and the braking
2 force is fully activated. Each of the intermediate notches 228 are positioned to receive the
3 plunger head 226 when the actuating lever 142 is rotated to positions between the upright,
4 vertical position and the horizontal position creating various levers of braking force. The spring
5 biased plunger head 226 presses into the notches 228 with sufficient force to maintain the
6 rotated position of the actuating lever 142 until sufficient manual force is exerted on the
7 actuating lever 142.

8 Although this alternative embodiment of the invention incorporates the use of notches
9 228 on the inner surface of the actuating lever 142 positioned to receive a plunger head 226, it
10 is anticipate that other means of maintaining the actuating lever 142 in a rotated position could
11 be used. For instance, it is anticipated that a ratcheting or gear mechanism could be used for
12 that purpose.

13 The operation of this alternative embodiment of the invention is identical to the
14 operation of the preferred embodiment with the exception of the use of notches 228 and
15 plunger assembly 220. These additional elements allow the user to rotate the actuating lever
16 142 into varying positions to exert a varying braking force on the disks 22. The combination of
17 the plunger assembly 220 and the notches 228 allow the user to release the actuating lever 142
18 and maintain the desired braking force and thereby allowing the user to keep both hands on the

1 wheels for steering or for other purposes while braking.

2 Although the invention has been described with reference to specific embodiments, this
3 description is not meant to be construed in a limited sense. Various modifications of the
4 disclosed embodiments, as well as alternative embodiments of the inventions will become
5 apparent to persons skilled in the art upon the reference to the description of the invention. It
6 is, therefore, contemplated that the appended claims will cover such modifications that fall
7 within the scope of the invention.